${\it MUS420}$ Introduction to Linear State Space Models

Julius O. Smith III (jos@ccrma.stanford.edu)
Center for Computer Research in Music and Acoustics (CCRMA)
Department of Music, Stanford University
Stanford, California 94305

February 5, 2019

Outline

- State Space Models
- Linear State Space Formulation
- Markov Parameters (Impulse Response)
- Transfer Function
- Difference Equations to State Space Models
- Similarity Transformations
- Modal Representation (Diagonalization)
- Matlab Examples

1

State-Space History

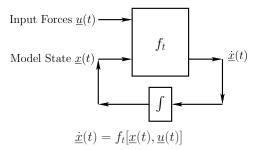
- 1. Classic *phase-space* in physics (Gibbs 1901) System state = point in *position-momentum space*
- 2. Digital computer (1950s)
- 3. Finite State Machines (Mealy and Moore, 1960s)
- 4. Finite Automata
- 5. State-Space Models of Linear Systems
- 6. Reference:

Linear system theory: The state space approach

L.A. Zadeh and C.A. Desoer Krieger, 1979

State Space Models

Equations of motion for any physical system may be conveniently formulated in terms of its state $\underline{x}(t)$:



where

x(t) = state of the system at time t

 $\underline{u}(t) = \text{vector of } external inputs (typically driving forces)$

 $f_t=$ general function mapping the current state $\underline{x}(t)$ and inputs $\underline{u}(t)$ to the state time-derivative $\dot{\underline{x}}(t)$

- ullet The function f_t may be time-varying, in general
- This potentially nonlinear time-varying model is extremely general (but causal)
- Even the human brain can be modeled in this form

2

Key Property of State Vector

The key property of the state vector $\underline{x}(t)$ in the state space formulation is that it completely determines the system at time t

- ullet Future states depend only on the current state $\underline{x}(t)$ and on any inputs u(t) at time t and beyond
- ullet All past states and the entire input history are "summarized" by the current state x(t)
- ullet State $\underline{x}(t)$ includes all "memory" of the system

3

4

Force-Driven Mass Example

Consider f=ma for the force-driven mass:

- ullet Since the mass m is constant, we can use momentum $p(t)=m\,v(t)$ in place of velocity (more fundamental, since momentum is conserved)
- $x(t_0)$ and $p(t_0)$ (or $v(t_0)$) define the *state* of the mass m at time t_0
- In the absence of external forces f(t), all future states are *predictable* from the state at time t_0 :

$$p(t) = p(t_0)$$
 (conservation of momentum)

$$x(t) = x(t_0) + \frac{1}{m} \int_{t_0}^{t} p(\tau) d\tau, \quad t \ge t_0$$

ullet External forces f(t) drive the state to arbitrary points in state space:

$$p(t) = p(t_0) + \int_{t_0}^{t} f(\tau) d\tau, \quad t \ge t_0, \quad p(t) \stackrel{\triangle}{=} m v(t)$$

$$x(t) = x(t_0) + \frac{1}{m} \int_{t_0}^t p(\tau) d\tau, \quad t \ge t_0$$

5

Numerical Integration

Recall the general state-space model in continuous time:

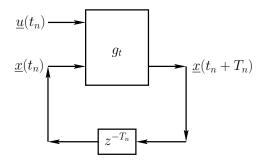
$$\underline{\dot{x}}(t) = f_t[\underline{x}(t), \underline{u}(t)]$$

An approximate discrete-time numerical solution is

$$\underline{x}(t_n + T_n) = \underline{x}(t_n) + T_n f_{t_n}[\underline{x}(t_n), \underline{u}(t_n)]$$

for $n = 0, 1, 2, \dots$ (Forward Euler)

Let $g_{t_n}[\underline{x}(t_n),\underline{u}(t_n)] \stackrel{\Delta}{=} \underline{x}(t_n) + T_n f_{t_n}[\underline{x}(t_n),\underline{u}(t_n)]$:

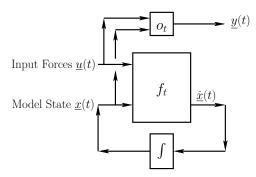


- This is a simple example of numerical integration for solving the ODE
- ODE can be nonlinear and/or time-varying
- \bullet The sampling interval T_n may be fixed or adaptive

Forming Outputs

Any system *output* is some function of the state, and possibly the input (directly):

$$y(t) \stackrel{\Delta}{=} o_t[\underline{x}(t), \underline{u}(t)]$$



Usually the output is a *linear combination* of state variables and possibly the current input:

$$y(t) \stackrel{\Delta}{=} \mathbf{C}\underline{x}(t) + \mathbf{D}\underline{u}(t)$$

where \boldsymbol{C} and \boldsymbol{D} are constant matrices of linear-combination coefficients

6

State Definition

We need a *state variable* for the amplitude of each *physical degree of freedom*

Examples:

• Ideal Mass:

$$\mathsf{Energy} = \frac{1}{2} m v^2 \ \Rightarrow \ \mathsf{state} \ \mathsf{variable} = v(t)$$

Note that in 3D we get three state variables (v_x,v_u,v_z)

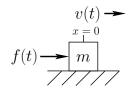
• Ideal Spring:

$$\mathsf{Energy} = \frac{1}{2}kx^2 \ \Rightarrow \ \mathsf{state} \ \mathsf{variable} = x(t)$$

- Inductor: Analogous to mass, so current
- Capacitor: Analogous to spring, so charge (or voltage = charge/capacitance)
- Resistors and dashpots need no state variables assigned—they are stateless (no "memory")

State-Space Model of a Force-Driven Mass

For the simple example of a mass m driven by external force f along the x axis:



- There is only one energy-storage element (the mass), and it stores energy in the form of kinetic energy
- Therefore, we should choose the state variable to be velocity $v=\dot{x}$ (or momentum $p=mv=m\dot{x}$)
- ullet Newton's f=ma readily gives the state-space formulation:

 $\dot{v} = \frac{1}{m}f$

or

• This is a first-order system (no vector needed)

9

Force-Driven Mass Reconsidered and Dismissed

• Position x does not affect stored energy

$$E_m = \frac{1}{2} m v^2$$

- ullet Velocity v(t) is the only energy-storing degree of freedom
- ullet Only velocity v(t) is needed as a state variable
- ullet The initial position x(0) can be kept "on the side" to enable computation of the complete state in position-momentum space:

$$x(t) = x(0) + \int_0^t v(\tau) d\tau$$

- In other words, the position can be derived from the velocity history without knowing the force history
- \bullet Note that the external force f(t) can only drive $\dot{v}(t).$ It cannot drive $\dot{x}(t)$ directly:

$$\left[\begin{array}{c} \dot{x}(t) \\ \dot{v}(t) \end{array} \right] \; = \; \left[\begin{array}{c} 0 \; 1 \\ 0 \; 0 \end{array} \right] \left[\begin{array}{c} x(t) \\ v(t) \end{array} \right] + \left[\begin{array}{c} 0 \\ 1/m \end{array} \right] f(t)$$

Force-Driven Mass Reconsidered

Why not include position x(t) as well as velocity v(t) in the state-space model for the force-driven mass?

$$\begin{bmatrix} \dot{x}(t) \\ \dot{v}(t) \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} x(t) \\ v(t) \end{bmatrix} + \begin{bmatrix} 0 \\ 1/m \end{bmatrix} f(t)$$

We might expect this because we know from before that the complete physical state of a mass consists of its velocity v and position x!

10

State Variable Summary

- State variable = physical amplitude for some energy-storing degree of freedom
- Mechanical Systems:

State variable for each

- ideal spring (linear or rotational)
- point mass (or moment of inertia)

times the number of dimensions in which it can move

• RLC Electric Circuits:

State variable for each capacitor and inductor

• In Discrete-Time:

State variable for each unit-sample delay

• Continuous- or Discrete-Time:

Dimensionality of state-space = order of the system (LTI systems)

Discrete-Time Linear State Space Models

For linear, time-invariant systems, a discrete-time state-space model looks like a vector first-order finite-difference model:

$$\underline{x}(n+1) = \mathbf{A}\underline{x}(n) + \mathbf{B}\underline{u}(n)$$

 $y(n) = \mathbf{C}\underline{x}(n) + \mathbf{D}\underline{u}(n)$

where

- ullet $\underline{x}(n) \in \mathbb{R}^N = \mathit{state vector}$ at time n
- $\underline{u}(n) = p \times 1$ vector of inputs
- $y(n) = q \times 1$ output vector
- $\mathbf{A} = N \times N$ state transition matrix
- $\mathbf{B} = N \times p$ input coefficient matrix
- $\mathbf{C} = q \times N$ output coefficient matrix
- $\mathbf{D} = q \times p$ direct path coefficient matrix

The state-space representation is especially powerful for

- multi-input, multi-output (MIMO) linear systems
- *time-varying* linear systems (every matrix can have a time subscript n)

13

Zero-State Impulse Response (Markov Parameters)

Thus, the "impulse response" of the state-space model can be summarized as

$$\mathbf{h}(n) = \begin{cases} \mathbf{D}, & n = 0 \\ \mathbf{C}\mathbf{A}^{n-1}\mathbf{B}, & n > 0 \end{cases}$$

- ullet Initial state $\underline{x}(0)$ assumed to be $\underline{0}$
- Input "impulse" is $u = \mathbf{I}_n \delta(n) = \mathsf{diag}(\delta(n), \dots, \delta(n))$
- \bullet Each "impulse-response sample" $\mathbf{h}(n)$ is a $p\times q$ matrix, in general
- ullet The impulse-response terms ${f C}{f A}^n{f B}$ for $n\geq 0$ are called ${\it Markov\ parameters}$

Zero-State Impulse Response (Markov Parameters)

Linear State-Space Model:

$$\underline{y}(n) = \mathbf{C}\underline{x}(n) + \mathbf{D}\underline{u}(n)$$
$$\underline{x}(n+1) = \mathbf{A}\underline{x}(n) + \mathbf{B}\underline{u}(n)$$

The zero-state *impulse response* of a state-space model is easily found by direct calculation: Let $\underline{x}(0) \stackrel{\Delta}{=} \underline{0}$ and $u = \mathbf{I}_n \delta(n) = \mathrm{diag}(\delta(n), \dots, \delta(n))$. Then

$$\mathbf{h}(0) = \mathbf{C}x(0)\mathbf{B} + \mathbf{D}\mathbf{I}_{p}\delta(0) = \mathbf{D}$$

$$x(1) = \mathbf{A} x(0) + \mathbf{B} \mathbf{I}_n \delta(0) = \mathbf{B}$$

$$\mathbf{h}(1) = \mathbf{CB}$$

$$\underline{x}(2) = \mathbf{A}\underline{x}(1) + \mathbf{B}\delta(1) = \mathbf{A}\mathbf{B}$$

$$\mathbf{h}(2) = \mathbf{C}\mathbf{A}\mathbf{B}$$

$$\underline{x}(3) = \mathbf{A} \underline{x}(1) + \mathbf{B} \delta(1) = \mathbf{A}^2 \mathbf{B}$$

$$\mathbf{h}(3) = \mathbf{C}\mathbf{A}^2\mathbf{B}$$

$$\mathbf{h}(n) = \boxed{\mathbf{C}\mathbf{A}^{n-1}\mathbf{B}}, \quad n > 0$$

14

Linear State-Space Model Transfer Function

• Recall the linear state-space model:

$$y(n) = \mathbf{C} \underline{x}(n) + \mathbf{D} \underline{u}(n)$$

$$\underline{x}(n+1) = \mathbf{A}\underline{x}(n) + \mathbf{B}\underline{u}(n)$$

and its "impulse response"

$$\mathbf{h}(n) = \begin{cases} \mathbf{D}, & n = 0 \\ \mathbf{C}\mathbf{A}^{n-1}\mathbf{B}, & n > 0 \end{cases}$$

 The transfer function is the z transform of the impulse response:

$$\mathbf{H}(z) \stackrel{\Delta}{=} \sum_{n=0}^{\infty} \mathbf{h}(n) z^{-n} = \mathbf{D} + \sum_{n=1}^{\infty} \left(\mathbf{C} \mathbf{A}^{n-1} \mathbf{B} \right) z^{-n}$$
$$= \mathbf{D} + z^{-1} \mathbf{C} \left[\sum_{n=0}^{\infty} \left(z^{-1} \mathbf{A} \right)^{n} \right] \mathbf{B}$$

The closed-form sum of a matrix geometric series gives

$$\mathbf{H}(z) = \mathbf{D} + \mathbf{C} (z\mathbf{I} - \mathbf{A})^{-1} \mathbf{B}$$

(a $p \times q$ matrix of rational polynomials in z)

- If there are p inputs and q outputs, then $\mathbf{H}(z)$ is a $p \times q$ transfer-function matrix (or "matrix transfer function")
- Given transfer-function coefficients, many digital filter realizations are possible (different computing structures)

Example:
$$(p = 3, q = 2)$$

$$\mathbf{H}(z) = \begin{bmatrix} z^{-1} & \frac{1-z^{-1}}{1-0.5z^{-1}} & 1+z^{-1} \\ \frac{2+3z^{-1}}{1-0.1z^{-1}} & \frac{1+z^{-1}}{1-z^{-1}} & \frac{(1-z^{-1})^2}{(1-0.1z^{-1})(1-0.2z^{-1})} \end{bmatrix}$$

17

Initial-Condition Response

Going back to the time domain, we have the linear discrete-time state-space model

$$y(n) = \mathbf{C} \underline{x}(n) + \mathbf{D} \underline{u}(n)$$

$$x(n+1) = \mathbf{A} x(n) + \mathbf{B} u(n)$$

and its "impulse response"

$$\mathbf{h}(n) \ = \ \left\{ \begin{array}{l} \mathbf{D}, & n = 0 \\ \mathbf{C}\mathbf{A}^{n-1}\mathbf{B}, & n > 0 \end{array} \right.$$

Given zero inputs and initial state $\underline{x}(0) \neq \underline{0}$, we get

$$y_{..}(n) = \mathbf{C}\mathbf{A}^{n}\underline{x}(0), \quad n = 0, 1, 2, \dots$$

By superposition (for LTI systems), the complete response of a linear system is given by the sum of its forced response (such as the impulse response) and its initial-condition response

System Poles

Above, we found the transfer function to be

$$\mathbf{H}(z) = \mathbf{D} + \mathbf{C} (z\mathbf{I} - \mathbf{A})^{-1} \mathbf{B}$$

The poles of $\mathbf{H}(z)$ are the same as those of

$$H_p(z) \stackrel{\Delta}{=} (z\mathbf{I} - \mathbf{A})^{-1}$$

By *Cramer's rule* for matrix inversion, the denominator polynomial for $(z\mathbf{I} - \mathbf{A})^{-1}$ is given by the *determinant*:

$$d(z) \stackrel{\Delta}{=} |z\mathbf{I} - \mathbf{A}|$$

where $|\mathbf{Q}|$ denotes the *determinant* of the square matrix \mathbf{Q} (also written as $\det(\mathbf{Q})$.)

- In linear algebra, the polynomial $d(z) = |z\mathbf{I} \mathbf{A}|$ is called the *characteristic polynomial* for the matrix \mathbf{A}
- ullet The roots of the characteristic polynomial are called the *eigenvalues* of ${f A}$
- Thus, the *eigenvalues* of the state transition matrix A are the system *poles*
- Each mode of vibration gives rise to a pole pair

18

Difference Equation to State Space Form

A digital filter is often specified by its *difference equation* (Direct Form I). Second-order example:

$$y(n)=u(n)+2u(n-1)+3u(n-2)-\frac{1}{2}y(n-1)-\frac{1}{3}y(n-2)$$

Every *n*th order *difference equation* can be reformulated as a *first order vector* difference equation called the "state space" (or "state variable") representation:

$$\underline{x}(n+1) = \mathbf{A} \underline{x}(n) + \mathbf{B} u(n)$$

 $y(n) = \mathbf{C} \underline{x}(n) + \mathbf{D} u(n)$

For the above example, we have, as we'll show,

$$\mathbf{A} \stackrel{\Delta}{=} \begin{bmatrix} -\frac{1}{2} & -\frac{1}{3} \\ 1 & 0 \end{bmatrix} \quad \text{(state transition matrix)}$$

$$\mathbf{B} \stackrel{\Delta}{=} \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$
 (matrix routing input to state variables)

$$\mathbf{C} \stackrel{\Delta}{=} \begin{bmatrix} 3/2 \\ 8/3 \end{bmatrix}$$
 (output linear-combination matrix)

$$\mathbf{D} \stackrel{\Delta}{=} 1$$
 (direct feedforward coefficient)

Converting to State-Space Form by Hand

1. First, determine the filter transfer function $\mathbf{H}(z)$. In the example, the transfer function can be written, by inspection, as

$$\mathbf{H}(z) = \frac{1 + 2z^{-1} + 3z^{-2}}{1 + \frac{1}{2}z^{-1} + \frac{1}{3}z^{-2}}$$

2. If $\mathbf{h}(0) \neq 0$, we must "pull out" the parallel delay-free path:

$$\mathbf{H}(z) = d_0 + \frac{b_1 z^{-1} + b_2 z^{-2}}{1 + \frac{1}{2} z^{-1} + \frac{1}{2} z^{-2}}$$

Obtaining a common denominator and equating numerator coefficients yields

$$d_0 = 1$$

$$b_1 = 2 - \frac{1}{2} = \frac{3}{2}$$

$$b_2 = 3 - \frac{1}{3} = \frac{8}{3}$$

The same result is obtained using long or synthetic division

21

Matlab Conversion from Direct-Form to State-Space Form

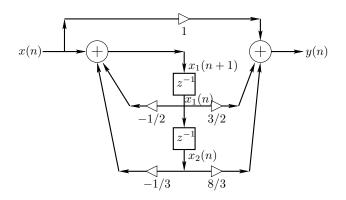
Matlab has extensive support for state-space models, such as

- tf2ss transfer-function to state-space conversion
- ss2tf state-space to transfer-function conversion

Note that these utilities are documented primarily for continuous-time systems, but they are also used for discrete-time systems.

Let's repeat the previous example using Matlab:

3. Next, draw the strictly causal part in *direct form II*, as shown below:



It is important that the filter representation be canonical with respect to delay, i.e., the number of delay elements equals the order of the filter

- 4. Assign a state variable to the output of each delay element (see figure)
- 5. Write down the state-space representation by inspection. (Try it and compare to answer above.)

22

Previous Example Using Matlab

>> num = [1 2 3]; % transfer function numerator
>> den = [1 1/2 1/3]; % denominator coefficients
>> [A,B,C,D] = tf2ss(num,den)

$$C = 1.5000 2.6667$$

$$D = 1$$

$$>> [N,D] = ss2tf(A,B,C,D)$$

$$N = 1.0000$$
 2.0000 3.0000

$$D = 1.0000 \quad 0.5000 \quad 0.3333$$

Matlab Documentation

The tf2ss and ss2tf functions are documented at http://www.mathworks.com/access/helpdesk/help/toolbox/signal/tf2ss.shtml as well as within Matlab itself (e.g., help tf2ss).

Related Signal Processing Toolbox functions include

- tf2sos Convert digital filter transfer function parameters to second-order sections form.
- sos2ss Convert second-order filter sections to state-space form.
- tf2zp Convert transfer function filter parameters to zero-pole-gain form.
- zp2ss Convert zero-pole-gain filter parameters to state-space form.

25

We can now write

$$\underline{\tilde{x}}(n+1) = \underline{\tilde{\mathbf{A}}}\underline{\tilde{x}}(n) + \underline{\tilde{\mathbf{B}}}\underline{u}(n)
y(n) = \underline{\tilde{\mathbf{C}}}\underline{\tilde{x}}(n) + \underline{\mathbf{D}}\underline{u}(n)$$

The transformed system describes the *same system* in new state-variable coordinates

Let's verify that the transfer function has not changed:

$$\begin{split} \tilde{\mathbf{H}}(z) &= \tilde{\mathbf{D}} + \tilde{\mathbf{C}}(z\mathbf{I} - \tilde{\mathbf{A}})^{-1}\tilde{\mathbf{B}} \\ &= \mathbf{D} + (\mathbf{C}\mathbf{E}) \left(z\mathbf{I} - \mathbf{E}^{-1}\mathbf{A}\mathbf{E}\right)^{-1} (\mathbf{E}^{-1}\mathbf{B}) \\ &= \mathbf{D} + \mathbf{C} \left[\mathbf{E} \left(z\mathbf{I} - \mathbf{E}^{-1}\mathbf{A}\mathbf{E}\right) \mathbf{E}^{-1}\right]^{-1}\mathbf{B} \\ &= \mathbf{D} + \mathbf{C} \left(z\mathbf{I} - \mathbf{A}\right)^{-1}\mathbf{B} = \mathbf{H}(z) \end{split}$$

- ullet Since the eigenvalues of A are the poles of the system, it follows that the eigenvalues of $\dot{A}=E^{-1}AE$ are the same. In other words, eigenvalues are unaffected by a similarity transformation.
- The transformed Markov parameters, $\tilde{\mathbf{C}}\tilde{\mathbf{A}}^n\tilde{\mathbf{B}}$, are also unchanged since they are given by the inverse z transform of the transfer function $\tilde{\mathbf{H}}(z)$. However, it is also easy to show this by direct calculation.

Similarity Transformations

A similarity transformation of a state-space system is a linear change of state variable coordinates:

$$\underline{x}(n) \stackrel{\Delta}{=} \mathbf{E} \underline{\tilde{x}}(n)$$

where

- x(n) = original state vector
- $\underline{\tilde{x}}(n) = \text{state vector in } \textit{new coordinates}$
- ullet E = any *invertible* (one-to-one) matrix (linear transformation)

Substituting $\underline{x}(n) = \mathbf{E}\underline{\tilde{x}}(n)$ gives

$$\mathbf{E}\underline{\tilde{x}}(n+1) = \mathbf{A} \mathbf{E}\underline{\tilde{x}}(n) + \mathbf{B}\underline{u}(n)$$
$$y(n) = \mathbf{C}\mathbf{E}\tilde{x}(n) + \mathbf{D}u(n)$$

Premultiplying the first equation above by \mathbf{E}^{-1} gives

$$\begin{array}{ll} \underline{\tilde{x}}(n+1) \ = \ \left(\mathbf{E}^{-1}\mathbf{A}\mathbf{E}\right)\underline{\tilde{x}}(n) + \left(\mathbf{E}^{-1}\mathbf{B}\right)\underline{u}(n) \\ \underline{y}(n) \ = \ \left(\mathbf{C}\mathbf{E}\right)\underline{\tilde{x}}(n) + \mathbf{D}\underline{u}(n) \end{array}$$

Define the transformed system matrices by

$$\begin{split} \tilde{\mathbf{A}} &= \mathbf{E}^{-1} \mathbf{A} \mathbf{E} \\ \tilde{\mathbf{B}} &= \mathbf{E}^{-1} \mathbf{B} \\ \tilde{\mathbf{C}} &= \mathbf{C} \mathbf{E} \\ \tilde{\mathbf{D}} &= \mathbf{D} \end{split}$$

26

State Space Modal Representation

Diagonal state transition matrix = modal representation:

$$\begin{bmatrix} x_1(n+1) \\ x_2(n+1) \\ \vdots \\ x_{N-1}(n+1) \\ x_N(n+1) \end{bmatrix} = \begin{bmatrix} \lambda_1 & 0 & 0 & \cdots & 0 \\ 0 & \lambda_2 & 0 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & 0 & \lambda_{N-1} & 0 \\ 0 & 0 & 0 & 0 & \lambda_N \end{bmatrix} \begin{bmatrix} x_1(n) \\ x_2(n) \\ \vdots \\ x_{N-1}(n) \\ x_N(n) \end{bmatrix} + \begin{bmatrix} b_1 \\ b_2 \\ \vdots \\ b_{N-1} \\ b_N \end{bmatrix} u(n)$$

$$y(n) = \mathbf{C}x(n) + \mathbf{D}u(n)$$

(always possible when there are no repeated poles)

The N complex modes are decoupled:

$$x_{1}(n+1) = \lambda_{1}x_{1}(n) + b_{1}u(n)$$

$$x_{2}(n+1) = \lambda_{2}x_{2}(n) + b_{2}u(n)$$

$$\vdots$$

$$x_{N}(n+1) = \lambda_{N}x_{N}(n) + b_{N}u(n)$$

$$y(n) = c_{1}x_{1}(n) + c_{2}x_{2}(n) + \dots + c_{N}x_{N}(n) + \mathbf{D}u(n)$$

That is, the diagonal state-space system consists of N parallel one-pole systems:

$$\mathbf{H}(z) = \mathbf{C}(z\mathbf{I} - \mathbf{A})^{-1}\mathbf{B} + \mathbf{D}$$
$$= \mathbf{D} + \sum_{i=1}^{N} \frac{c_i b_i z^{-1}}{1 - \lambda_i z^{-1}}$$

27

Finding the (Diagonalized) Modal Representation

The *i*th eigenvector \underline{e}_i of a matrix \mathbf{A} has the defining property

$$\mathbf{A}\underline{e}_i = \lambda_i \underline{e}_i$$
,

where λ_i is the associated *eigenvalue*. Thus, the eigenvector \underline{e}_i is *invariant* under the linear transformation \mathbf{A} to within a (generally complex) scale factor λ_i .

An $N \times N$ matrix ${\bf A}$ typically has N eigenvectors. Let's make a similarity-transformation matrix ${\bf E}$ out of the N eigenvectors:

$$\mathbf{E} = \left[\underline{e}_1 \ \underline{e}_2 \ \cdots \ \underline{e}_N \right]$$

Then we have

$$\mathbf{AE} = \begin{bmatrix} \lambda_1 \underline{e}_1 & \lambda_2 \underline{e}_2 & \cdots & \lambda_N \underline{e}_N \end{bmatrix} \stackrel{\Delta}{=} \mathbf{E} \mathbf{\Lambda}$$

where $\Lambda \stackrel{\triangle}{=} \operatorname{diag}(\underline{\lambda})$ is a diagonal matrix having $\underline{\lambda} \stackrel{\triangle}{=} \left[\begin{array}{ccc} \lambda_1 & \lambda_2 & \cdots & \lambda_N \end{array} \right]^T$ along its diagonal. Premultiplying by \mathbf{E}^{-1} gives

$$\mathbf{E}^{-1}\mathbf{A}\mathbf{E} = \mathbf{\Lambda}$$

Thus, $\mathbf{E} = \left[\ \underline{e}_1 \ \underline{e}_2 \ \cdots \ \underline{e}_N \ \right]$ is a similarity transformation that diagonalizes the system.

¹When there are repeated eigenvalues, there may be only one linearly independent eigenvector for the repeated group. We will not consider this case and refer the interested reader to a Web search on "generalized eigenvectors," e.g., http://en.wikipedia.org/wiki/Generalized_eigenvector.

20

or, in vector notation,

$$x(n+1) = \mathbf{A} x(n)$$

The poles of the system are given by the eigenvalues of \mathbf{A} , which are the roots of its characteristic polynomial. That is, we solve

$$|\lambda_i \mathbf{I} - \mathbf{A}| = 0$$

for λ_i , $i = 1, 2, \dots, N$, or, for our N = 2 problem,

$$0 = \begin{vmatrix} \lambda_i - c & 1 - c \\ -c - 1 & \lambda_i - c \end{vmatrix} = (\lambda_i - c)^2 + (1 - c)(1 + c) = \lambda_i^2 - 2\lambda_i c + 1$$

Using the quadratic formula, the two solutions are found to be

$$\lambda_i = c \pm \sqrt{c^2 - 1} = c \pm j\sqrt{1 - c^2}$$

Defining $c = \cos(\theta)$, we obtain the simple formula

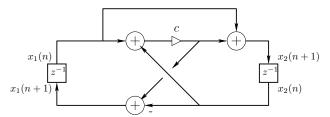
$$\lambda_i = \cos(\theta) \pm j \sin(\theta) = e^{\pm j\theta}$$

It is now clear that the system is a real sinusoidal oscillator for $-1 \leq c \leq 1$, oscillating at normalized radian frequency $\omega_c T \stackrel{\scriptscriptstyle \Delta}{=} \theta \stackrel{\scriptscriptstyle \Delta}{=} \arccos(c) \in [-\pi,\pi].$

We determined the frequency of oscillation $\omega_c T$ from the eigenvalues λ_i of \mathbf{A} . To study this system further, we can diagonalize \mathbf{A} . For that we need the eigenvectors as well as the eigenvalues.

State-Space Analysis Example: The Digital Waveguide Oscillator

Let's use state-space analysis to determine the frequency of oscillation of the following system:



The second-order digital waveguide oscillator.

Note the assignments of unit-delay *outputs* to state variables $x_1(n)$ and $x_2(n)$.

We have

$$x_1(n+1) = c[x_1(n) + x_2(n)] - x_2(n) = c \, x_1(n) + (c-1)x_2(n)$$
 and

$$x_2(n+1) = x_1(n) + c[x_1(n) + x_2(n)] = (1+c)x_1(n) + cx_2(n)$$

In matrix form, the state transition can be written as

$$\begin{bmatrix} x_1(n+1) \\ x_2(n+1) \end{bmatrix} = \underbrace{\begin{bmatrix} c & c-1 \\ c+1 & c \end{bmatrix}}_{\mathbf{A}} \begin{bmatrix} x_1(n) \\ x_2(n) \end{bmatrix}$$

Eigenstructure of A

The defining property of the eigenvectors \underline{e}_i and eigenvalues λ_i of \mathbf{A} is the relation

$$\mathbf{A}\underline{e_i} = \lambda_i \underline{e_i}, \quad i = 1, 2,$$

which expands to

$$\begin{bmatrix} c & c-1 \\ c+1 & c \end{bmatrix} \begin{bmatrix} 1 \\ \eta_i \end{bmatrix} = \begin{bmatrix} \lambda_i \\ \lambda_i \eta_i \end{bmatrix}.$$

- ullet The first element of \underline{e}_i is normalized arbitrarily to 1
- We have two equations in two unknowns λ_i and η_i :

$$c + \eta_i(c - 1) = \lambda_i$$

$$(1 + c) + c\eta_i = \lambda_i\eta_i$$

(We already know λ_i from above, but this analysis will find them by a different method.)

• Substitute the first into the second to eliminate λ_i :

$$1 + c + c\eta_i = [c + \eta_i(c - 1)]\eta_i = c\eta_i + \eta_i^2(c - 1)$$

$$\Rightarrow 1 + c = \eta_i^2(c - 1)$$

$$\Rightarrow \eta_i = \pm \sqrt{\frac{c + 1}{c - 1}}$$

• We have found both eigenvectors:

$$\underline{e}_1 \, = \, \left[\begin{array}{c} 1 \\ \eta \end{array} \right], \quad \underline{e}_2 = \left[\begin{array}{c} 1 \\ -\eta \end{array} \right], \quad \text{where } \eta \stackrel{\Delta}{=} \sqrt{\frac{c+1}{c-1}}$$

They are linearly independent provided $\eta \neq 0 \Leftrightarrow c \neq -1$ and finite provided $c \neq 1$.

• The eigenvalues are then

$$\lambda_i = c + \eta_i(c-1) = c \pm \sqrt{\frac{c+1}{c-1}(c-1)^2} = c \pm \sqrt{c^2 - 1}$$

 \bullet Assuming |c| < 1, they can be written as

$$\lambda_i = c \pm j\sqrt{1-c^2}$$

- With $c \in (-1,1)$, define $\theta = \arccos(c)$, i.e., $c \stackrel{\Delta}{=} \cos(\theta)$ and $\sqrt{1-c^2} = \sin(\theta)$.
- The eigenvalues become

$$\lambda_1 = c + j\sqrt{1 - c^2} = \cos(\theta) + j\sin(\theta) = e^{j\theta}$$

$$\lambda_2 = c - j\sqrt{1 - c^2} = \cos(\theta) - j\sin(\theta) = e^{-j\theta}$$

as expected.

We again found the explicit formula for the frequency of oscillation:

$$\omega_c = \frac{\theta}{T} = f_s \arccos(c),$$

33

We have two natural choices of output which are the state variables $x_1(n)$ and $x_2(n)$, corresponding to the choices C = [1, 0] and C = [0, 1]:

$$y_1(n) \stackrel{\Delta}{=} x_1(n) = [1, 0] \underline{x}(n)$$

 $y_2(n) \stackrel{\Delta}{=} x_2(n) = [0, 1] x(n)$

Thus, a convenient choice of the system ${\bf C}$ matrix is the 2×2 identity matrix.

For the diagonalized system we obtain

$$\tilde{\mathbf{A}} = \mathbf{E}^{-1}\mathbf{A}\mathbf{E} = \begin{bmatrix} e^{j\theta} & 0 \\ 0 & e^{-j\theta} \end{bmatrix}$$

$$\tilde{\mathbf{B}} = \mathbf{E}^{-1}\mathbf{B} = \mathbf{0}$$

$$\tilde{\mathbf{C}} = \mathbf{C}\mathbf{E} = \mathbf{E} = \begin{bmatrix} 1 & 1 \\ \eta & -\eta \end{bmatrix}$$

$$\tilde{\mathbf{D}} = 0$$

where $\theta = \arccos(c)$ and $\eta = \sqrt{\frac{c+1}{c-1}}$ as derived above.

We may now view our state-output signals in terms of

where f_s denotes the sampling rate. Or,

$$c = \cos(\omega_c T)$$

The coefficient range $c \in (-1,1)$ corresponds to frequencies $f \in (-f_s/2,f_s/2)$.

We have shown that the example system oscillates sinusoidally at any desired digital frequency ω_c when $c=\cos(\omega_c T)$, where T denotes the sampling interval.

The Diagonalized Example System

We can now diagonalize our system using the similarity transformation

$$\mathbf{E} = \left[\ \underline{e}_1 \ \underline{e}_2 \ \right] = \left[\ \begin{matrix} 1 & 1 \\ \eta & -\eta \end{matrix} \right]$$

where
$$\eta = \sqrt{\frac{c+1}{c-1}}$$
.

We have only been working with the state-transition matrix ${\bf A}$ up to now.

The system has no inputs so it must be excited by initial conditions (although we could easily define one or two inputs that sum into the delay elements).

34

the modal representation:

$$y_1(n) = [1, 0]\underline{x}(n) = [1, 0] \begin{bmatrix} 1 & 1 \\ \eta & -\eta \end{bmatrix} \underline{\tilde{x}}(n)$$

$$= [1, 1]\underline{\tilde{x}}(n) = \lambda_1^n \, \tilde{x}_1(0) + \lambda_2^n \, \tilde{x}_2(0)$$

$$y_2(n) = [0, 1]\underline{x}(n) = [0, 1] \begin{bmatrix} 1 & 1 \\ \eta & -\eta \end{bmatrix} \underline{\tilde{x}}(n)$$

$$= [\eta, -\eta]\underline{\tilde{x}}(n) = \eta \lambda_1^n \tilde{x}_1(0) - \eta \lambda_2^n \, \tilde{x}_2(0)$$

The output signal from the first state variable $x_1(n)$ is

$$y_1(n) = \lambda_1^n \tilde{x}_1(0) + \lambda_2^n \tilde{x}_2(0) = e^{j\omega_c nT} \tilde{x}_1(0) + e^{-j\omega_c nT} \tilde{x}_2(0)$$

The initial condition $\underline{x}(0) = [1,0]^T$ corresponds to modal initial state

$$\underline{\tilde{x}}(0) = \mathbf{E}^{-1} \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \frac{-1}{2e} \begin{bmatrix} -e & -1 \\ -e & 1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} 1/2 \\ 1/2 \end{bmatrix}$$

For this initialization, the output y_1 from the first state variable x_1 is simply

$$y_1(n) = \frac{e^{j\omega_c nT} + e^{-j\omega_c nT}}{2} = \boxed{\cos(\omega_c nT)}$$

Similarly $y_2(n)$ is proportional to $\sin(\omega_c nT)$ ("phase quadrature" output), with amplitude η .