## State Space Models

MUS420
Introduction to Linear State Space Models

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## Outline

- State Space Models
- Linear State Space Formulation
- Markov Parameters (Impulse Response)
- Transfer Function
- Difference Equations to State Space Models
- Similarity Transformations
- Modal Representation (Diagonalization)
- Matlab Examples


## State-Space History

1. Classic phase-space in physics (Gibbs 1901) System state $=$ point in position-momentum space
2. Digital computer (1950s)
3. Finite State Machines (Mealy and Moore, 1960s)
4. Finite Automata
5. State-Space Models of Linear Systems
6. Reference:

Linear system theory: The state space approach
L.A. Zadeh and C.A. Desoer

Krieger, 1979

Equations of motion for any physical system may be conveniently formulated in terms of its state $\underline{x}(t)$ :

where
$\underline{x}(t)=$ state of the system at time $t$
$\underline{u}(t)=$ vector of external inputs (typically driving forces)
$f_{t}=$ general function mapping the current state $\underline{x}(t)$ and
inputs $\underline{u}(t)$ to the state time-derivative $\underline{\dot{x}}(t)$

- The function $f_{t}$ may be time-varying, in general
- This potentially nonlinear time-varying model is extremely general (but causal)
- Even the human brain can be modeled in this form


## Key Property of State Vector

The key property of the state vector $\underline{x}(t)$ in the state space formulation is that it completely determines the system at time $t$

- Future states depend only on the current state $\underline{x}(t)$ and on any inputs $\underline{u}(t)$ at time $t$ and beyond
- All past states and the entire input history are "summarized" by the current state $\underline{x}(t)$
- State $\underline{x}(t)$ includes all "memory" of the system


## Force-Driven Mass Example

Consider $f=m a$ for the force-driven mass:

- Since the mass $m$ is constant, we can use momentum $p(t)=m v(t)$ in place of velocity (more fundamental, since momentum is conserved)
- $x\left(t_{0}\right)$ and $p\left(t_{0}\right)$ (or $v\left(t_{0}\right)$ ) define the state of the mass $m$ at time $t_{0}$
- In the absence of external forces $f(t)$, all future states are predictable from the state at time $t_{0}$ :

$$
\begin{aligned}
& p(t)=p\left(t_{0}\right) \quad \text { (conservation of momentum) } \\
& x(t)=x\left(t_{0}\right)+\frac{1}{m} \int_{t_{0}}^{t} p(\tau) d \tau, \quad t \geq t_{0}
\end{aligned}
$$

- External forces $f(t)$ drive the state to arbitrary points in state space:

$$
\begin{aligned}
& p(t)=p\left(t_{0}\right)+\int_{t_{0}}^{t} f(\tau) d \tau, \quad t \geq t_{0}, \quad p(t) \triangleq m v(t) \\
& x(t)=x\left(t_{0}\right)+\frac{1}{m} \int_{t_{0}}^{t} p(\tau) d \tau, \quad t \geq t_{0}
\end{aligned}
$$

## Numerical Integration

Recall the general state-space model in continuous time:

$$
\underline{\dot{x}}(t)=f_{t}[\underline{x}(t), \underline{u}(t)]
$$

An approximate discrete-time numerical solution is

$$
\underline{x}\left(t_{n}+T_{n}\right)=\underline{x}\left(t_{n}\right)+T_{n} f_{t_{n}}\left[\underline{x}\left(t_{n}\right), \underline{u}\left(t_{n}\right)\right]
$$

for $n=0,1,2, \ldots$ (Forward Euler)
Let $\left.g_{t_{n}}\left[\underline{x}\left(t_{n}\right), \underline{u}\left(t_{n}\right)\right] \triangleq \underline{x}\left(t_{n}\right)+T_{n} f_{t_{n}} \underline{x}\left(t_{n}\right), \underline{u}\left(t_{n}\right)\right]$ :


- This is a simple example of numerical integration for solving the ODE
- ODE can be nonlinear and/or time-varying
- The sampling interval $T_{n}$ may be fixed or adaptive

Any system output is some function of the state, and possibly the input (directly):

$$
\underline{y}(t) \triangleq o_{t}[\underline{x}(t), \underline{u}(t)]
$$



Usually the output is a linear combination of state variables and possibly the current input:

$$
\underline{y}(t) \triangleq \mathbf{C} \underline{x}(t)+\mathbf{D} \underline{u}(t)
$$

where $\mathbf{C}$ and $\mathbf{D}$ are constant matrices of linear-combination coefficients

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## State Definition

We need a state variable for the amplitude of each physical degree of freedom

## Examples:

- Ideal Mass:

$$
\text { Energy }=\frac{1}{2} m v^{2} \Rightarrow \text { state variable }=v(t)
$$

Note that in 3D we get three state variables $\left(v_{x}, v_{y}, v_{z}\right)$

- Ideal Spring:

$$
\text { Energy }=\frac{1}{2} k x^{2} \Rightarrow \text { state variable }=x(t)
$$

- Inductor: Analogous to mass, so current
- Capacitor: Analogous to spring, so charge (or voltage $=$ charge/capacitance)
- Resistors and dashpots need no state variables assigned-they are stateless (no "memory")


## State-Space Model of a Force-Driven Mass

For the simple example of a mass $m$ driven by external force $f$ along the $x$ axis:


- There is only one energy-storage element (the mass), and it stores energy in the form of kinetic energy
- Therefore, we should choose the state variable to be velocity $v=\dot{x}$ (or momentum $p=m v=m \dot{x}$ )
- Newton's $f=m a$ readily gives the state-space formulation:

$$
\dot{v}=\frac{1}{m} f
$$

or

$$
\dot{p}=f
$$

- This is a first-order system (no vector needed)


## Force-Driven Mass Reconsidered and Dismissed

- Position $x$ does not affect stored energy

$$
E_{m}=\frac{1}{2} m v^{2}
$$

- Velocity $v(t)$ is the only energy-storing degree of freedom
- Only velocity $v(t)$ is needed as a state variable
- The initial position $x(0)$ can be kept "on the side" to enable computation of the complete state in position-momentum space:

$$
x(t)=x(0)+\int_{0}^{t} v(\tau) d \tau
$$

- In other words, the position can be derived from the velocity history without knowing the force history
- Note that the external force $f(t)$ can only drive $\dot{v}(t)$. It cannot drive $\dot{x}(t)$ directly:

$$
\left[\begin{array}{c}
\dot{x}(t) \\
\dot{v}(t)
\end{array}\right]=\left[\begin{array}{ll}
0 & 1 \\
0 & 0
\end{array}\right]\left[\begin{array}{l}
x(t) \\
v(t)
\end{array}\right]+\left[\begin{array}{c}
0 \\
1 / m
\end{array}\right] f(t)
$$

Why not include position $x(t)$ as well as velocity $v(t)$ in the state-space model for the force-driven mass?

$$
\left[\begin{array}{c}
\dot{x}(t) \\
\dot{v}(t)
\end{array}\right]=\left[\begin{array}{ll}
0 & 1 \\
0 & 0
\end{array}\right]\left[\begin{array}{c}
x(t) \\
v(t)
\end{array}\right]+\left[\begin{array}{c}
0 \\
1 / m
\end{array}\right] f(t)
$$

We might expect this because we know from before that the complete physical state of a mass consists of its velocity $v$ and position $x$ !

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## State Variable Summary

- State variable $=$ physical amplitude for some energy-storing degree of freedom
- Mechanical Systems:

State variable for each

- ideal spring (linear or rotational)
- point mass (or moment of inertia)
times the number of dimensions in which it can move


## - RLC Electric Circuits:

State variable for each capacitor and inductor

- In Discrete-Time:

State variable for each unit-sample delay

## - Continuous- or Discrete-Time:

Dimensionality of state-space $=$ order of the system (LTI systems)

## Discrete-Time Linear State Space Models

For linear, time-invariant systems, a discrete-time state-space model looks like a vector first-order finite-difference model:

$$
\begin{aligned}
\underline{x}(n+1) & =\mathbf{A} \underline{x}(n)+\mathbf{B} \underline{u}(n) \\
\underline{y}(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} \underline{u}(n)
\end{aligned}
$$

where

- $\underline{x}(n) \in \mathbb{R}^{N}=$ state vector at time $n$
- $\underline{u}(n)=p \times 1$ vector of inputs
- $\underline{y}(n)=q \times 1$ output vector
- $\mathbf{A}=N \times N$ state transition matrix
- $\mathbf{B}=N \times p$ input coefficient matrix
- $\mathbf{C}=q \times N$ output coefficient matrix
- $\mathbf{D}=q \times p$ direct path coefficient matrix

The state-space representation is especially powerful for

- multi-input, multi-output (MIMO) linear systems
- time-varying linear systems (every matrix can have a time subscript $n$ )


## Zero-State Impulse Response (Markov Parameters)

Thus, the "impulse response" of the state-space model can be summarized as

$$
\mathbf{h}(n)= \begin{cases}\mathbf{D}, & n=0 \\ \mathbf{C A}^{n-1} \mathbf{B}, & n>0\end{cases}
$$

- Initial state $\underline{x}(0)$ assumed to be $\underline{0}$
- Input "impulse" is $\underline{u}=\mathbf{I}_{p} \delta(n)=\operatorname{diag}(\delta(n), \ldots, \delta(n))$
- Each "impulse-response sample" $\mathbf{h}(n)$ is a $p \times q$ matrix, in general
- The impulse-response terms $\mathbf{C A}^{n} \mathbf{B}$ for $n \geq 0$ are called Markov parameters


## Zero-State Impulse Response (Markov Parameters)

Linear State-Space Model:

$$
\begin{aligned}
\underline{y}(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} \underline{u}(n) \\
\underline{x}(n+1) & =\mathbf{A} \underline{x}(n)+\mathbf{B} \underline{u}(n)
\end{aligned}
$$

The zero-state impulse response of a state-space model is easily found by direct calculation: Let $\underline{x}(0) \triangleq \underline{0}$ and $\underline{u}=\mathbf{I}_{p} \delta(n)=\operatorname{diag}(\delta(n), \ldots, \delta(n))$. Then

$$
\begin{aligned}
\mathbf{h}(0) & =\mathbf{C} \underline{x}(0) \mathbf{B}+\mathbf{D I}_{p} \delta(0)=\mathbf{D} \\
\underline{x}(1) & =\mathbf{A} \underline{x}(0)+\mathbf{B} \mathbf{I}_{p} \delta(0)=\mathbf{B} \\
\mathbf{h}(1) & =\mathbf{C B} \\
\underline{x}(2) & =\mathbf{A} \underline{x}(1)+\mathbf{B} \delta(1)=\mathbf{A B} \\
\mathbf{h}(2) & =\mathbf{C A B} \\
\underline{x}(3) & =\mathbf{A} \underline{x}(1)+\mathbf{B} \delta(1)=\mathbf{A}^{2} \mathbf{B} \\
\mathbf{h}(3) & =\mathbf{C A}^{2} \mathbf{B} \\
& \vdots \\
\mathbf{h}(n) & =\mathbf{C A}^{n-1} \mathbf{B}, \quad n>0
\end{aligned}
$$

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## Linear State-Space Model Transfer Function

- Recall the linear state-space model:

$$
\begin{aligned}
\underline{y}(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} \underline{u}(n) \\
\underline{x}(n+1) & =\mathbf{A} \underline{x}(n)+\mathbf{B} \underline{u}(n)
\end{aligned}
$$

and its "impulse response"

$$
\mathbf{h}(n)= \begin{cases}\mathbf{D}, & n=0 \\ \mathbf{C A}^{n-1} \mathbf{B}, & n>0\end{cases}
$$

- The transfer function is the $z$ transform of the impulse response:

$$
\begin{aligned}
\mathbf{H}(z) \triangleq \sum_{n=0}^{\infty} \mathbf{h}(n) z^{-n} & =\mathbf{D}+\sum_{n=1}^{\infty}\left(\mathbf{C A}^{n-1} \mathbf{B}\right) z^{-n} \\
& =\mathbf{D}+z^{-1} \mathbf{C}\left[\sum_{n=0}^{\infty}\left(z^{-1} \mathbf{A}\right)^{n}\right] \mathbf{B}
\end{aligned}
$$

The closed-form sum of a matrix geometric series gives

$$
\mathbf{H}(z)=\mathbf{D}+\mathbf{C}(z \mathbf{I}-\mathbf{A})^{-1} \mathbf{B}
$$

(a $p \times q$ matrix of rational polynomials in $z$ )

- If there are $p$ inputs and $q$ outputs, then $\mathbf{H}(z)$ is a $p \times q$ transfer-function matrix (or "matrix transfer function")
- Given transfer-function coefficients, many digital filter realizations are possible (different computing structures)

$$
\text { Example: }(p=3, q=2)
$$

$\mathbf{H}(z)=\left[\begin{array}{ccc}z^{-1} & \frac{1-z^{-1}}{1-0.5 z^{-1}} & 1+z^{-1} \\ \frac{2+3 z^{-1}}{1-0.1 z^{-1}} & \frac{1+z^{-1}}{1-z^{-1}} & \frac{\left(1-z^{-1}\right)^{2}}{\left(1-0.1 z^{-1}\right)\left(1-0.2 z^{-1}\right)}\end{array}\right]$

## Initial-Condition Response

Going back to the time domain, we have the linear discrete-time state-space model

$$
\begin{aligned}
\underline{y}(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} \underline{u}(n) \\
\underline{x}(n+1) & =\mathbf{A} \underline{x}(n)+\mathbf{B} \underline{u}(n)
\end{aligned}
$$

and its "impulse response"

$$
\mathbf{h}(n)= \begin{cases}\mathbf{D}, & n=0 \\ \mathbf{C A}^{n-1} \mathbf{B}, & n>0\end{cases}
$$

Given zero inputs and initial state $\underline{x}(0) \neq \underline{0}$, we get

$$
\underline{y}_{x}(n)=\mathbf{C A}^{n} \underline{x}(0), \quad n=0,1,2, \ldots
$$

By superposition (for LTI systems), the complete response of a linear system is given by the sum of its forced response (such as the impulse response) and its initial-condition response

## System Poles

Above, we found the transfer function to be

$$
\mathbf{H}(z)=\mathbf{D}+\mathbf{C}(z \mathbf{I}-\mathbf{A})^{-1} \mathbf{B}
$$

The poles of $\mathbf{H}(z)$ are the same as those of

$$
H_{p}(z) \triangleq(z \mathbf{I}-\mathbf{A})^{-1}
$$

By Cramer's rule for matrix inversion, the denominator polynomial for $(z \mathbf{I}-\mathbf{A})^{-1}$ is given by the determinant:

$$
d(z) \triangleq|z \mathbf{I}-\mathbf{A}|
$$

where $|\mathbf{Q}|$ denotes the determinant of the square matrix $\mathbf{Q}$ (also written as $\operatorname{det}(\mathbf{Q})$.)

- In linear algebra, the polynomial $d(z)=|z \mathbf{I}-\mathbf{A}|$ is called the characteristic polynomial for the matrix A
- The roots of the characteristic polynomial are called the eigenvalues of $\mathbf{A}$
- Thus, the eigenvalues of the state transition matrix A are the system poles
- Each mode of vibration gives rise to a pole pair

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## Difference Equation to State Space Form

A digital filter is often specified by its difference equation (Direct Form I). Second-order example:
$y(n)=u(n)+2 u(n-1)+3 u(n-2)-\frac{1}{2} y(n-1)-\frac{1}{3} y(n-2)$
Every $n$th order difference equation can be reformulated as a first order vector difference equation called the "state space" (or "state variable") representation:

$$
\begin{aligned}
\underline{x}(n+1) & =\mathbf{A} \underline{x}(n)+\mathbf{B} u(n) \\
y(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} u(n)
\end{aligned}
$$

For the above example, we have, as we'll show,
$\mathbf{A} \triangleq\left[\begin{array}{cc}-\frac{1}{2} & -\frac{1}{3} \\ 1 & 0\end{array}\right] \quad$ (state transition matrix)
$\mathbf{B} \triangleq\left[\begin{array}{l}1 \\ 0\end{array}\right] \quad$ (matrix routing input to state variables)
$\mathbf{C} \triangleq\left[\begin{array}{l}3 / 2 \\ 8 / 3\end{array}\right] \quad$ (output linear-combination matrix)
D $\triangleq 1 \quad$ (direct feedforward coefficient)

## Converting to State-Space Form by Hand

1. First, determine the filter transfer function $\mathbf{H}(z)$. In the example, the transfer function can be written, by inspection, as

$$
\mathbf{H}(z)=\frac{1+2 z^{-1}+3 z^{-2}}{1+\frac{1}{2} z^{-1}+\frac{1}{3} z^{-2}}
$$

2. If $\mathbf{h}(0) \neq 0$, we must "pull out" the parallel delay-free path:

$$
\mathbf{H}(z)=d_{0}+\frac{b_{1} z^{-1}+b_{2} z^{-2}}{1+\frac{1}{2} z^{-1}+\frac{1}{3} z^{-2}}
$$

Obtaining a common denominator and equating numerator coefficients yields

$$
\begin{aligned}
d_{0} & =1 \\
b_{1} & =2-\frac{1}{2}=\frac{3}{2} \\
b_{2} & =3-\frac{1}{3}=\frac{8}{3}
\end{aligned}
$$

The same result is obtained using long or synthetic division

## Matlab Conversion from Direct-Form to State-Space Form

## Matlab has extensive support for state-space models,

 such as- tf2ss - transfer-function to state-space conversion
- ss2tf - state-space to transfer-function conversion

Note that these utilities are documented primarily for continuous-time systems, but they are also used for discrete-time systems.
Let's repeat the previous example using Matlab:
3. Next, draw the strictly causal part in direct form II, as shown below:


It is important that the filter representation be canonical with respect to delay, i.e., the number of delay elements equals the order of the filter
4. Assign a state variable to the output of each delay element (see figure)
5. Write down the state-space representation by inspection. (Try it and compare to answer above.)

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## Previous Example Using Matlab

```
>> num = [1 2 3]; % transfer function numerator
>> den = [1 1/2 1/3]; % denominator coefficients
>> [A,B,C,D] = tf2ss(num,den)
A =
    -0.5000 -0.3333
    1.0000
B =
    1
    0
C = 1.5000 2.6667
D = 1
>> [N,D] = ss2tf(A,B,C,D)
N = 1.0000 2.0000 3.0000
D = 1.0000
    0.5000
    0.3333
```


## Matlab Documentation

The tf 2 ss and ss 2 tf functions are documented at http://www.mathworks. com/access/helpdesk/help/toolbox/signal/tf2ss. shtm1 as well as within Matlab itself (e.g., help tf2ss).

Related Signal Processing Toolbox functions include
-tf2sos - Convert digital filter transfer function parameters to second-order sections form.

- sos2ss - Convert second-order filter sections to state-space form.
- tf2zp - Convert transfer function filter parameters to zero-pole-gain form.
- zp2ss - Convert zero-pole-gain filter parameters to state-space form.


## Similarity Transformations

A similarity transformation of a state-space system is a linear change of state variable coordinates:

$$
\underline{x}(n) \triangleq \mathbf{E} \underline{\tilde{x}}(n)
$$

where

- $\underline{x}(n)=$ original state vector
- $\underline{\tilde{x}}(n)=$ state vector in new coordinates
- $\mathrm{E}=$ any invertible (one-to-one) matrix (linear transformation)
Substituting $\underline{x}(n)=\mathbf{E} \underline{\tilde{x}}(n)$ gives

$$
\begin{aligned}
\mathbf{E} \underline{\tilde{x}}(n+1) & =\mathbf{A} \mathbf{E} \underline{\tilde{x}}(n)+\mathbf{B} \underline{u}(n) \\
\underline{y}(n) & =\mathbf{C} \mathbf{E} \underline{\tilde{x}}(n)+\mathbf{D} \underline{u}(n)
\end{aligned}
$$

Premultiplying the first equation above by $\mathbf{E}^{-1}$ gives

$$
\begin{aligned}
\underline{\tilde{x}}(n+1) & =\left(\mathbf{E}^{-1} \mathbf{A E}\right) \underline{\tilde{x}}(n)+\left(\mathbf{E}^{-1} \mathbf{B}\right) \underline{u}(n) \\
\underline{y}(n) & =(\mathbf{C E}) \underline{\tilde{x}}(n)+\mathbf{D} \underline{u}(n)
\end{aligned}
$$

Define the transformed system matrices by

$$
\begin{aligned}
& \tilde{\mathbf{A}}=\mathbf{E}^{-1} \mathbf{A E} \\
& \tilde{\mathbf{B}}=\mathbf{E}^{-1} \mathbf{B} \\
& \tilde{\mathbf{C}}=\mathbf{C E} \\
& \tilde{\mathrm{D}}=\mathbf{D}
\end{aligned}
$$

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We can now write

$$
\begin{aligned}
\underline{\tilde{x}}(n+1) & =\tilde{\mathbf{A}} \tilde{\tilde{x}}(n)+\tilde{\mathbf{B}} \underline{u}(n) \\
\underline{y}(n) & =\tilde{\mathbf{C}} \underline{\tilde{x}}(n)+\mathbf{D} \underline{u}(n)
\end{aligned}
$$

The transformed system describes the same system in new state-variable coordinates
Let's verify that the transfer function has not changed:

$$
\begin{aligned}
\tilde{\mathbf{H}}(z) & =\tilde{\mathbf{D}}+\tilde{\mathbf{C}}(z \mathbf{I}-\tilde{\mathbf{A}})^{-1} \tilde{\mathbf{B}} \\
& =\mathbf{D}+(\mathbf{C E})\left(z \mathbf{I}-\mathbf{E}^{-1} \mathbf{A E}\right)^{-1}\left(\mathbf{E}^{-1} \mathbf{B}\right) \\
& =\mathbf{D}+\mathbf{C}\left[\mathbf{E}\left(z \mathbf{I}-\mathbf{E}^{-1} \mathbf{A E}\right) \mathbf{E}^{-1}\right]^{-1} \mathbf{B} \\
& =\mathbf{D}+\mathbf{C}(z \mathbf{I}-\mathbf{A})^{-1} \mathbf{B}=\mathbf{H}(z)
\end{aligned}
$$

- Since the eigenvalues of $\mathbf{A}$ are the poles of the system, it follows that the eigenvalues of $\tilde{\mathbf{A}}=\mathbf{E}^{-1} \mathbf{A E}$ are the same. In other words, eigenvalues are unaffected by a similarity transformation.
- The transformed Markov parameters, $\tilde{\mathbf{C}} \tilde{\mathbf{A}}^{n} \tilde{\mathbf{B}}$, are also unchanged since they are given by the inverse $z$ transform of the transfer function $\tilde{\mathbf{H}}(z)$. However, it is also easy to show this by direct calculation.


## State Space Modal Representation

Diagonal state transition matrix $=$ modal representation:

$$
\begin{aligned}
{\left[\begin{array}{c}
x_{1}(n+1) \\
x_{2}(n+1) \\
\vdots \\
x_{N-1}(n+1) \\
x_{N}(n+1)
\end{array}\right] } & =\left[\begin{array}{ccccc}
\lambda_{1} & 0 & 0 & \cdots & 0 \\
0 & \lambda_{2} & 0 & \cdots & 0 \\
\vdots & \vdots & \ddots & \vdots & \vdots \\
0 & 0 & 0 & \lambda_{N-1} & 0 \\
0 & 0 & 0 & 0 & \lambda_{N}
\end{array}\right]\left[\begin{array}{c}
x_{1}(n) \\
x_{2}(n) \\
\vdots \\
x_{N-1}(n) \\
x_{N}(n)
\end{array}\right]+\left[\begin{array}{c}
b_{1} \\
b_{2} \\
\vdots \\
b_{N-1} \\
b_{N}
\end{array}\right] u(n) \\
y(n) & =\mathbf{C} \underline{x}(n)+\mathbf{D} u(n)
\end{aligned}
$$

(always possible when there are no repeated poles)
The $N$ complex modes are decoupled:

$$
\begin{aligned}
x_{1}(n+1) & =\lambda_{1} x_{1}(n)+b_{1} u(n) \\
x_{2}(n+1) & =\lambda_{2} x_{2}(n)+b_{2} u(n) \\
& \vdots \\
x_{N}(n+1) & =\lambda_{N} x_{N}(n)+b_{N} u(n) \\
y(n) & =c_{1} x_{1}(n)+c_{2} x_{2}(n)+\cdots+c_{N} x_{N}(n)+\mathbf{D} u(n)
\end{aligned}
$$

That is, the diagonal state-space system consists of $N$ parallel one-pole systems:

$$
\begin{aligned}
\mathbf{H}(z) & =\mathbf{C}(z \mathbf{I}-\mathbf{A})^{-1} \mathbf{B}+\mathbf{D} \\
& =\mathbf{D + \sum _ { i = 1 } ^ { N } \frac { c _ { i } b _ { i } z ^ { - 1 } } { 1 - \lambda _ { i } z ^ { - 1 } }}
\end{aligned}
$$

## Finding the (Diagonalized) Modal Representation

The $i$ th eigenvector $\underline{e}_{i}$ of a matrix $\mathbf{A}$ has the defining property

$$
\mathbf{A} \underline{e}_{i}=\lambda_{i} \underline{e}_{i}
$$

where $\lambda_{i}$ is the associated eigenvalue. Thus, the eigenvector $\underline{e}_{i}$ is invariant under the linear transformation A to within a (generally complex) scale factor $\lambda_{i}$.
An $N \times N$ matrix A typically has $N$ eigenvectors. 1 Let's make a similarity-transformation matrix $\mathbf{E}$ out of the $N$ eigenvectors:

$$
\mathbf{E}=\left[\begin{array}{llll}
\underline{e}_{1} & \underline{e}_{2} & \cdots & \underline{e}_{N}
\end{array}\right]
$$

Then we have

$$
\mathbf{A E}=\left[\begin{array}{llll}
\lambda_{1} \underline{e}_{1} & \lambda_{2} \underline{e}_{2} & \cdots & \lambda_{N} \underline{e}_{N}
\end{array}\right] \triangleq \mathbf{E} \mathbf{E}
$$

where $\Lambda \triangleq \operatorname{diag}(\underline{\lambda})$ is a diagonal matrix having $\underline{\lambda} \triangleq\left[\begin{array}{llll}\lambda_{1} & \lambda_{2} & \cdots & \lambda_{N}\end{array}\right]^{T}$ along its diagonal.
Premultiplying by $\mathbf{E}^{-1}$ gives

$$
\mathbf{E}^{-1} \mathbf{A E}=\boldsymbol{\Lambda}
$$

Thus, $\mathbf{E}=\left[\begin{array}{llll}\underline{e}_{1} & \underline{e}_{2} & \cdots & \underline{e}_{N}\end{array}\right]$ is a similarity transformation that diagonalizes the system.

[^0]or, in vector notation,
$$
\underline{x}(n+1)=\mathbf{A} \underline{x}(n)
$$

The poles of the system are given by the eigenvalues of A, which are the roots of its characteristic polynomial. That is, we solve

$$
\left|\lambda_{i} \mathbf{I}-\mathbf{A}\right|=0
$$

for $\lambda_{i}, i=1,2, \ldots, N$, or, for our $N=2$ problem,
$0=\left|\begin{array}{cc}\lambda_{i}-c & 1-c \\ -c-1 & \lambda_{i}-c\end{array}\right|=\left(\lambda_{i}-c\right)^{2}+(1-c)(1+c)=\lambda_{i}{ }^{2}-2 \lambda_{i} c+1$
Using the quadratic formula, the two solutions are found to be

$$
\lambda_{i}=c \pm \sqrt{c^{2}-1}=c \pm j \sqrt{1-c^{2}}
$$

Defining $c=\cos (\theta)$, we obtain the simple formula

$$
\lambda_{i}=\cos (\theta) \pm j \sin (\theta)=e^{ \pm j \theta}
$$

It is now clear that the system is a real sinusoidal oscillator for $-1 \leq c \leq 1$, oscillating at normalized radian frequency $\omega_{c} T \triangleq \bar{\theta} \triangleq \arccos (c) \in[-\pi, \pi]$.
We determined the frequency of oscillation $\omega_{c} T$ from the eigenvalues $\lambda_{i}$ of $\mathbf{A}$. To study this system further, we can diagonalize A. For that we need the eigenvectors as well as the eigenvalues.

## State-Space Analysis Example: <br> The Digital Waveguide Oscillator

Let's use state-space analysis to determine the frequency of oscillation of the following system:


The second-order digital waveguide oscillator.
Note the assignments of unit-delay outputs to state variables $x_{1}(n)$ and $x_{2}(n)$.
We have
$x_{1}(n+1)=c\left[x_{1}(n)+x_{2}(n)\right]-x_{2}(n)=c x_{1}(n)+(c-1) x_{2}(n)$
and
$x_{2}(n+1)=x_{1}(n)+c\left[x_{1}(n)+x_{2}(n)\right]=(1+c) x_{1}(n)+c x_{2}(n)$
In matrix form, the state transition can be written as

$$
\left[\begin{array}{l}
x_{1}(n+1) \\
x_{2}(n+1)
\end{array}\right]=\underbrace{\left[\begin{array}{cc}
c & c-1 \\
c+1 & c
\end{array}\right]}_{\mathbf{A}}\left[\begin{array}{l}
x_{1}(n) \\
x_{2}(n)
\end{array}\right]
$$

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## Eigenstructure of $\mathbf{A}$

The defining property of the eigenvectors $\underline{e}_{i}$ and eigenvalues $\lambda_{i}$ of $\mathbf{A}$ is the relation

$$
\mathbf{A} \underline{e}_{i}=\lambda_{i} \underline{e}_{i}, \quad i=1,2
$$

which expands to

$$
\left[\begin{array}{cc}
c & c-1 \\
c+1 & c
\end{array}\right]\left[\begin{array}{c}
1 \\
\eta_{i}
\end{array}\right]=\left[\begin{array}{c}
\lambda_{i} \\
\lambda_{i} \eta_{i}
\end{array}\right]
$$

- The first element of $\underline{e}_{i}$ is normalized arbitrarily to 1
- We have two equations in two unknowns $\lambda_{i}$ and $\eta_{i}$ :

$$
\begin{aligned}
c+\eta_{i}(c-1) & =\lambda_{i} \\
(1+c)+c \eta_{i} & =\lambda_{i} \eta_{i}
\end{aligned}
$$

(We already know $\lambda_{i}$ from above, but this analysis will find them by a different method.)

- Substitute the first into the second to eliminate $\lambda_{i}$ :

$$
\begin{aligned}
1+c+c \eta_{i} & =\left[c+\eta_{i}(c-1)\right] \eta_{i}=c \eta_{i}+\eta_{i}^{2}(c-1) \\
\Rightarrow 1+c & =\eta_{i}^{2}(c-1) \\
\Rightarrow \eta_{i} & = \pm \sqrt{\frac{c+1}{c-1}}
\end{aligned}
$$

- We have found both eigenvectors:

$$
\underline{e}_{1}=\left[\begin{array}{l}
1 \\
\eta
\end{array}\right], \quad \underline{e}_{2}=\left[\begin{array}{c}
1 \\
-\eta
\end{array}\right], \quad \text { where } \eta \triangleq \sqrt{\frac{\Delta+1}{c-1}}
$$

They are linearly independent provided $\eta \neq 0 \Leftrightarrow c \neq-1$ and finite provided $c \neq 1$.

- The eigenvalues are then

$$
\lambda_{i}=c+\eta_{i}(c-1)=c \pm \sqrt{\frac{c+1}{c-1}(c-1)^{2}}=c \pm \sqrt{c^{2}-1}
$$

- Assuming $|c|<1$, they can be written as

$$
\lambda_{i}=c \pm j \sqrt{1-c^{2}}
$$

- With $c \in(-1,1)$, define $\theta=\arccos (c)$, i.e., $c \triangleq \cos (\theta)$ and $\sqrt{1-c^{2}}=\sin (\theta)$.
- The eigenvalues become

$$
\begin{aligned}
& \lambda_{1}=c+j \sqrt{1-c^{2}}=\cos (\theta)+j \sin (\theta)=e^{j \theta} \\
& \lambda_{2}=c-j \sqrt{1-c^{2}}=\cos (\theta)-j \sin (\theta)=e^{-j \theta}
\end{aligned}
$$

as expected.
We again found the explicit formula for the frequency of oscillation:

$$
\omega_{c}=\frac{\theta}{T}=f_{s} \arccos (c)
$$

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We have two natural choices of output which are the state variables $x_{1}(n)$ and $x_{2}(n)$, corresponding to the choices $\mathbf{C}=[1,0]$ and $\mathbf{C}=[0,1]$ :

$$
\begin{aligned}
& y_{1}(n) \triangleq x_{1}(n)=[1,0] \underline{x}(n) \\
& y_{2}(n) \triangleq x_{2}(n)=[0,1] \underline{x}(n)
\end{aligned}
$$

Thus, a convenient choice of the system $\mathbf{C}$ matrix is the $2 \times 2$ identity matrix.
For the diagonalized system we obtain

$$
\begin{aligned}
& \tilde{\mathbf{A}}=\mathbf{E}^{-1} \mathbf{A} \mathbf{E}=\left[\begin{array}{cc}
e^{j \theta} & 0 \\
0 & e^{-j \theta}
\end{array}\right] \\
& \tilde{\mathbf{B}}=\mathbf{E}^{-1} \mathbf{B}=\mathbf{0} \\
& \tilde{\mathbf{C}}=\mathbf{C E}=\mathbf{E}=\left[\begin{array}{cc}
1 & 1 \\
\eta & -\eta
\end{array}\right] \\
& \tilde{\mathbf{D}}=0
\end{aligned}
$$

where $\theta=\arccos (c)$ and $\eta=\sqrt{\frac{c+1}{c-1}}$ as derived above.
We may now view our state-output signals in terms of
where $f_{s}$ denotes the sampling rate. Or,

$$
c=\cos \left(\omega_{c} T\right)
$$

The coefficient range $c \in(-1,1)$ corresponds to frequencies $f \in\left(-f_{s} / 2, f_{s} / 2\right)$.

We have shown that the example system oscillates sinusoidally at any desired digital frequency $\omega_{c}$ when $c=\cos \left(\omega_{c} T\right)$, where $T$ denotes the sampling interval.

## The Diagonalized Example System

We can now diagonalize our system using the similarity transformation

$$
\mathbf{E}=\left[\underline{e}_{1} \underline{e}_{2}\right]=\left[\begin{array}{cc}
1 & 1 \\
\eta & -\eta
\end{array}\right]
$$

where $\eta=\sqrt{\frac{c+1}{c-1}}$.
We have only been working with the state-transition matrix A up to now.
The system has no inputs so it must be excited by initial conditions (although we could easily define one or two inputs that sum into the delay elements).

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the modal representation:

$$
\begin{aligned}
y_{1}(n) & =[1,0] \underline{x}(n)=[1,0]\left[\begin{array}{cc}
1 & 1 \\
\eta & -\eta
\end{array}\right] \underline{\underline{x}}(n) \\
& =[1,1] \underline{\underline{x}}(n)=\lambda_{1}^{n} \tilde{x}_{1}(0)+\lambda_{2}^{n} \tilde{x}_{2}(0) \\
y_{2}(n) & =[0,1] \underline{x}(n)=[0,1]\left[\begin{array}{cc}
1 & 1 \\
\eta & -\eta
\end{array}\right] \underline{\tilde{x}}(n) \\
& =[\eta,-\eta] \underline{\tilde{x}}(n)=\eta \lambda_{1}^{n} \tilde{x}_{1}(0)-\eta \lambda_{2}^{n} \tilde{x}_{2}(0)
\end{aligned}
$$

The output signal from the first state variable $x_{1}(n)$ is

$$
\begin{aligned}
y_{1}(n) & =\lambda_{1}^{n} \tilde{x}_{1}(0)+\lambda_{2}^{n} \tilde{x}_{2}(0) \\
& =e^{j \omega_{c} n T} \tilde{x}_{1}(0)+e^{-j \omega_{c} n T} \tilde{x}_{2}(0)
\end{aligned}
$$

The initial condition $\underline{x}(0)=[1,0]^{T}$ corresponds to modal initial state

$$
\underline{\tilde{x}}(0)=\mathbf{E}^{-1}\left[\begin{array}{l}
1 \\
0
\end{array}\right]=\frac{-1}{2 e}\left[\begin{array}{cc}
-e & -1 \\
-e & 1
\end{array}\right]\left[\begin{array}{l}
1 \\
0
\end{array}\right]=\left[\begin{array}{l}
1 / 2 \\
1 / 2
\end{array}\right]
$$

For this initialization, the output $y_{1}$ from the first state variable $x_{1}$ is simply

$$
y_{1}(n)=\frac{e^{j \omega_{c} n T}+e^{-j \omega_{c} n T}}{2}=\cos \left(\omega_{c} n T\right)
$$

Similarly $y_{2}(n)$ is proportional to $\sin \left(\omega_{c} n T\right)$
("phase quadrature" output), with amplitude $\eta$.


[^0]:    ${ }^{1}$ When there are repeated eigenvalues, there may be only one linearly independent eigenvector for the repeated group. We will not consider this case and refer the interested reader to a Web search on "generalized eigenvectors," e.g., http://en.wikipedia.org/wiki/Generalized_eigenvector.

